

Table of Contents

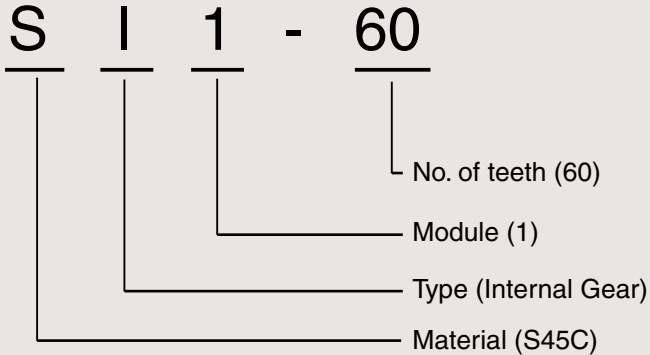
Special Characteristics, Points of Caution
in Selecting and Using Internal Gears..... page 148
SI Internal Gears page 150
SIR Ring Gears page 151

Catalog Number of KHK Stock Gears

The Catalog Number for KHK stock gears is based on the simple formula listed below.
Please order KHK gears by specifying the Catalog Numbers.

(Example)

Internal Gears



Material
S S45C

Type
I Internal Gears
IR Ring Gears

3

Internal Gears





Internal Gears

For Convenient Use in Various Purposes Such as Planetary Gear Drives and Segment Gears.



Characteristics

KHK stock internal gears are offered in modules 1 to 3 in 50 to 200 teeth. They can be used in many applications including planetary gear drives.

■ SI Internal Gears

SI internal gears can be combined with SS and SSA spur gears to make planetary gear drives.

■ SIR ring gears

SIR ring gears can be cut to make segment gears and corner racks.

■ Calculation of Bending Strength of Gears

| Item | Catalog No. | SI | SIR |
|--|-------------|---|-----|
| Formula <small>NOTE 1</small> | | Formula of spur and helical gears on bending strength(JGMA401-01) | |
| No. of teeth of mating gears | | 30 | |
| Rotation | | 100min ⁻¹ | |
| Durability | | Over 10 ⁷ cycles | |
| Impact from motor | | Uniform load | |
| Impact from load | | Uniform load | |
| Direction of load | | Bidirectional | |
| Allowable beam stress at root <small>σ_{Flim} NOTE 2</small> | | 12.67kgf/mm ² | |
| Safety factor S _F | | 1.2 | |



Selection Hints

Please select the most suitable products by carefully considering the characteristics of items and contents of the product tables. It is also important to read all applicable notes before the final selection. Use of catalog numbers when ordering will simplify and expedite the processing of your order.

1. Caution in Selecting the Mating Gears

Most KHK stock spur gears may be used as the mating gears except CP spur gears and gears with large numbers of teeth. When the difference in the numbers of teeth between an internal gear and its mating gear is small, involute interference, trochoid interference or trimming interference may occur.

2. Caution in Selecting Gears Based on Gear Strength

The gear strength values shown in the product pages were computed by assuming a certain application environment. Therefore, they should be used as reference only. We recommend that each user computes his own values by applying the actual usage conditions. The table on the right contains the assumptions established for these products in order to compute gear strengths.



■ Calculation of Surface Durability (Except where it is common with bending strength)

| Formula <small>NOTE 1</small> | Formula of spur and helical gears on surface durability (JGMA402-01) |
|--|--|
| Kinematic viscosity of lubricant | 100cSt(50°C) |
| Gear support | Symmetric support by bearings |
| Allowable Hertz stress <small>σ_{Hlim}</small> | 49kgf/mm ² |
| Safety factor S _H | 1.15 |

NOTE 1:The gear strength formula is based on JGMA (Japanese Gear Manufacturers' Association) The units for the rotational speed (min⁻¹) and the stress (kgf/mm²) are adjusted to the units needed in the formula.

NOTE 2:Since the load is bidirectional, the allowable bending stress at root σ_{Flim} calculated is set to 2/3 of the value.

3. Other Points to Consider in Selection Process

- ① There are various footnotes to the product pages under the headings of "NOTES". Please consider them carefully when selecting these products.
- ② There may be slight differences in color or shape of products shown in the photograph from the actual products.
- ③ KHK reserves the right to make changes in specifications and dimensions without notice.
- ④ KHK is ready to produce and supply custom order products. When you require specific gears different from KHK Stock Gears please contact our distributor for quotation. Also, please refer to page 16 "KHK Custom Order Products".



Application Hints

In order to use KHK stock internal gears safely, read the Application Hints carefully before proceeding. Also “1. Caution on Performing Secondary Operations”, “3. Notes on Starting Operations” and “4. Other Points to Consider in Applications” in the spur gear section should be consulted.

1. Point of Caution in Assembling

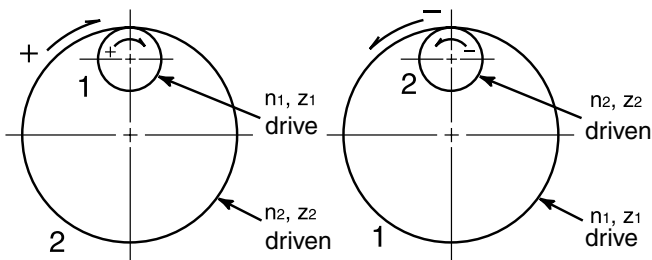
① KHK stock internal gears are designed to give the proper backlash when assembled using the center distance given by the formula below. The amount of backlash is given in the product table for each gear.

$$a = \frac{d_1 - d_2}{2}$$

where
 a = center distance
 d₁ = pitch diameter of pinion
 d₂ = pitch diameter of internal gear

② Note that the direction of rotation of the internal gear is different from that of two spur gears in mesh

Gear Ratio and Direction of Rotation



(a) Internal gear is driven (b) Internal gear drives

$$\text{Gear ratio } i = \frac{z_1}{z_2} = \frac{n_2}{n_1}$$

z = No. of teeth
 n = Rotational speed

③ Various types of interference and their symptoms and causes are tabulated below

| TYPE | SYMPTOMS | CAUSES |
|-----------------------|--|--|
| Involute interference | The tip of the internal gear digs into the root of the pinion. | Too few teeth on the pinion. |
| Trochoid interference | The exiting pinion tooth contacts the internal gear tooth. | Too little difference in number of teeth of the two gears. |
| Trimming interference | Pinion can slide in or out axially but cannot move radially. | Too little difference in number of teeth of the two gears. |

④ To use as a planetary gear drive, the following conditions must be satisfied.

Condition on number of teeth in planetary mechanism

Condition 1... $z_c = z_a + 2z_b$

Condition 2... $\frac{z_a + z_c}{N} = \text{Integer}$

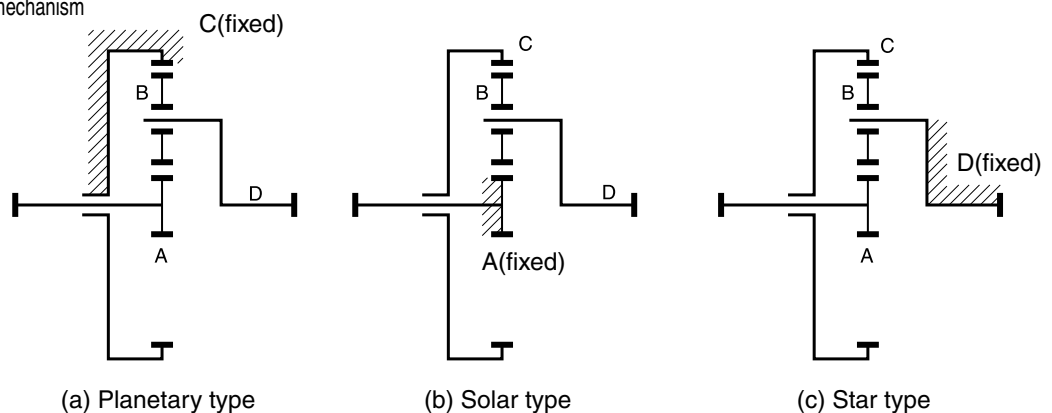
Condition 3... $z_b + 2 < (z_a + z_b) \sin \frac{180^\circ}{N}$

z_a : No. of teeth of sun gear
 z_b : No. of teeth of planet gears
 z_c : No. of teeth of internal gear
 N : No. of planet gears

Example of combinations

| No. of teeth of internal gear | No. of planet gears | No. of teeth of sun gear | No. of teeth of planet gears | Reduction ratio of planetary type | Reduction ratio of solar type | Reduction ratio of star type |
|-------------------------------|---------------------|--------------------------|------------------------------|-----------------------------------|-------------------------------|------------------------------|
| 50 | 3 | 16 | 17 | 1/4.125 | 1/1.32 | -1/3.125 |
| 80 | 3 | 16 | 32 | 1/6 | 1/1.2 | -1/5 |
| 80 | 3 | 40 | 20 | 1/3 | 1/1.5 | -1/2 |
| 100 | 3 | 20 | 40 | 1/6 | 1/1.2 | -1/5 |
| 100 | 3 | 50 | 25 | 1/3 | 1/1.5 | -1/2 |

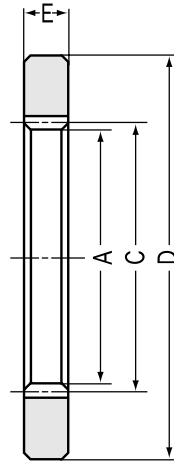
Types of Planetary gear reduction mechanism



(a) Planetary type

(b) Solar type

(c) Star type



T1 Shape

Specifications

| | |
|--|--|
| Precision | JIS N8 grade (JIS1 B1702-1: 1998) OLD JIS 4 grade (JIS B1702: 1976) |
| Gear teeth | Standard full depth |
| Pressure angle | 20° |
| Material | S45C |
| Heat treatment | — |
| Tooth hardness | Less than 194HB |
| Surface treatment | Black oxide |
| Tooth surface finishing | Cut |
| Datum reference surface for gear cutting | Outside diameter |
| Secondary Operations | Possible |

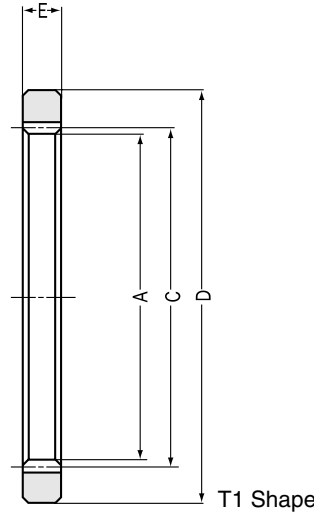
Module 1, 1.5, 2, 2.5, 3

| Catalog No. | Module | No. of teeth | Shape | Internal dia. | | Pitch dia. | Outside dia. | Face width | Allowable torque (N·m) NOTE 1 | | Allowable torque (kgf·m) | | Backlash (mm) NOTE 2 | Weight (kg) |
|------------------|-------------|--------------|-------|---------------|-----|------------|--------------|------------|-------------------------------|------|--------------------------|--------------------|----------------------|-------------|
| | | | | A | C | | | | D | E | Bending strength | Surface durability | | |
| SI1-60 | m1 | 60 | T1 | 58 | 60 | 90 | 10 | 30.0 | 5.95 | 3.06 | 0.61 | 0.11~0.24 | 0.28 | |
| SI1-80 | | 80 | T1 | 78 | 80 | 110 | 10 | 38.8 | 6.59 | 3.96 | 0.67 | 0.11~0.24 | 0.35 | |
| SI1-100 | | 100 | T1 | 98 | 100 | 130 | 10 | 47.8 | 7.64 | 4.87 | 0.78 | 0.13~0.27 | 0.42 | |
| SI1.5-50 | m1.5 | 50 | T1 | 72 | 75 | 115 | 15 | 87.1 | 20.9 | 8.88 | 2.13 | 0.13~0.29 | 0.71 | |
| SI1.5-60 | | 60 | T1 | 87 | 90 | 130 | 15 | 101 | 20.6 | 10.3 | 2.10 | 0.13~0.29 | 0.81 | |
| SI1.5-80 | | 80 | T1 | 117 | 120 | 160 | 15 | 131 | 23.3 | 13.4 | 2.38 | 0.13~0.29 | 1.00 | |
| SI1.5-100 | | 100 | T1 | 147 | 150 | 190 | 15 | 161 | 27.0 | 16.5 | 2.75 | 0.15~0.32 | 1.30 | |
| SI2-50 | m2 | 50 | T1 | 96 | 100 | 150 | 20 | 206 | 50.3 | 21.0 | 5.13 | 0.16~0.33 | 1.50 | |
| SI2-60 | | 60 | T1 | 116 | 120 | 170 | 20 | 240 | 50.5 | 24.5 | 5.15 | 0.16~0.33 | 1.80 | |
| SI2-80 | | 80 | T1 | 156 | 160 | 210 | 20 | 311 | 57.0 | 31.7 | 5.81 | 0.16~0.33 | 2.30 | |
| SI2-100 | | 100 | T1 | 196 | 200 | 250 | 20 | 382 | 65.7 | 39.0 | 6.70 | 0.17~0.37 | 2.80 | |
| SI2.5-50 | m2.5 | 50 | T1 | 120 | 125 | 185 | 25 | 403 | 101 | 41.1 | 10.3 | 0.17~0.37 | 2.70 | |
| SI2.5-60 | | 60 | T1 | 145 | 150 | 210 | 25 | 469 | 101 | 47.8 | 10.3 | 0.17~0.37 | 3.30 | |
| SI2.5-80 | | 80 | T1 | 195 | 200 | 260 | 25 | 607 | 114 | 61.9 | 11.6 | 0.17~0.37 | 4.10 | |
| SI3-50 | m3 | 50 | T1 | 144 | 150 | 220 | 30 | 697 | 178 | 71.0 | 18.1 | 0.19~0.41 | 4.80 | |
| SI3-60 | | 60 | T1 | 174 | 180 | 250 | 30 | 811 | 178 | 82.7 | 18.2 | 0.19~0.41 | 5.60 | |

CAUTION: Please check for the involute interference, trochoid interference and trimming interference prior to using internal gears.

NOTE 1: The allowable torques shown in the table are the calculated values according to the assumed usage conditions. Please see page 148 for more details.

NOTE 2: The backlash values shown in the table are the theoretical values of a 30 tooth SS spur gear in mesh with the internal gear.



Specifications

| | |
|--|---|
| Precision | JIS N9 grade (JIS B1702-1: 1998) OLD JIS 5 grade (JIS B1702: 1976) |
| Gear teeth | Standard full depth |
| Pressure angle | 20° |
| Material | S45C |
| Heat treatment | — |
| Tooth hardness | Less than 194HB |
| Surface treatment | Black oxide |
| Tooth surface finishing | Cut |
| Datum reference surface for gear cutting | Outside diameter |
| Secondary Operations | Possible |

Internal Gears
SIR

Module 2, 2.5, 3

| Catalog No. | Module | No. of teeth | Sharpe | Internal dia. | Pitch dia. | Outside dia. | Face width | Allowable torque (N · m) NOTE 1 | | Allowable torque (kgf · m) | | Backlash (mm) NOTE 2 | Weight (kg) |
|-------------------|-------------|--------------|--------|---------------|------------|--------------|------------|---------------------------------|--------------------|----------------------------|--------------------|----------------------|-------------|
| | | | | A | C | D | E | Bending strength | Surface durability | Bending strength | Surface durability | | |
| SIR2-120 | m2 | 120 | T1 | 236 | 240 | 286 | 20 | 413 | 68.8 | 42.1 | 7.02 | 0.17~0.37 | 3.00 |
| SIR2-200 | | 200 | T1 | 396 | 400 | 446 | 20 | 677 | 110 | 69.0 | 11.2 | 0.20~0.41 | 4.80 |
| SIR2.5-120 | m2.5 | 120 | T1 | 295 | 300 | 355 | 25 | 807 | 138 | 82.3 | 14.0 | 0.19~0.41 | 5.50 |
| SIR2.5-200 | | 200 | T1 | 495 | 500 | 555 | 25 | 1320 | 220 | 135 | 22.5 | 0.22~0.46 | 8.90 |
| SIR3-120 | m3 | 120 | T1 | 354 | 360 | 424 | 30 | 1390 | 244 | 142 | 24.9 | 0.22~0.45 | 10.0 |
| SIR3-160 | | 160 | T1 | 474 | 480 | 544 | 30 | 1840 | 315 | 188 | 32.1 | 0.22~0.45 | 12.1 |

CAUTION: Ring gears are susceptible to deformation, so careful handling is required.

NOTE 1: The allowable torques shown in the table are the calculated values according to the assumed usage conditions. Please see page 148 for more details.

NOTE 2: The backlash values shown in the table are the theoretical values of a 30 tooth SS spur gear in mesh with the internal gear.